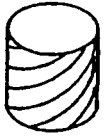


14.1-1



Imagine stiff fibers in rubber. Uniform  $\sigma_z$  on ends makes it unwind; i.e.  $v \neq 0$ . Can make  $v = 0$  by adding torque, but then (by statics)  $\tau_{\theta z} = 0$ . Need both  $v = 0$  and  $\tau_{\theta z} = 0$  for axisymmetric behavior.

14.2-1

|                            | <u>zero <math>\epsilon</math> energy</u> | <u>nonzero <math>\epsilon</math> energy</u> |
|----------------------------|--|---|
| (a) Plane , 1 Gauss pt.    | 1, 2, 3, 7, 8                            | 4, 5, 6                                     |
| (b) Plane , 4 Gauss pts.   | 1, 2, 3                                  | 4, 5, 6, 7, 8                               |
| (c) Axisym. , 1 Gauss pt.  | 1, 3, 7, 8                               | 2, 3, 4, 5                                  |
| (d) Axisym. , 4 Gauss pts. | 1  | 2 through 8                                 |

14.2-2

For a one-radian segment, with  $r = r_7 + J\xi$  and  $J = \frac{r_3 - r_4}{2}$ ,

$$\{\underline{r}_e\} = \int [N]^T \{\underline{\Phi}\} dS = \int_{-1}^1 \begin{Bmatrix} (\xi^2 - \xi)/2 \\ 1 - \xi^2 \\ (\xi^2 + \xi)/2 \end{Bmatrix} p r J d\xi$$

(a)  $p$  is constant:

$$\{\underline{r}_e\} = \frac{p J r_7}{3} \begin{Bmatrix} 1 \\ 4 \\ 1 \end{Bmatrix} + \frac{p J^2}{3} \begin{Bmatrix} -1 \\ 0 \\ 1 \end{Bmatrix}$$

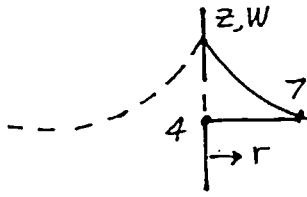
(b)

$$p = \frac{\xi^2 - \xi}{2} p_4 + (1 - \xi^2) p_7 + \frac{\xi^2 + \xi}{2} p_3$$

Algebra is straightforward but tedious

$$\{\underline{r}_e\} = \frac{r_7 J}{30} \begin{Bmatrix} 8p_4 + 4p_7 - 2p_3 \\ 4p_4 + 32p_7 + 4p_3 \\ -2p_4 + 4p_7 + 8p_3 \end{Bmatrix} + \frac{J^2}{30} \begin{Bmatrix} -6p_4 - 4p_7 \\ -4p_4 + 4p_3 \\ 4p_7 + 6p_3 \end{Bmatrix}$$

14.2-3



Implies  $\delta r_z$  dis-  
continuous across  
 $r=0$ ; not rea-  
sonable; not  
compatible.

Need  $\frac{dw}{dz} = 0$  at  $r=0$ , but need such a  
nodal d.o.f. to achieve it.

14.2-4

For compatibility,  $u=0$  at  $r=0$   
(no hole appears). Expand  $u$ :

$$u = c_1 r + c_2 z + c_3 r^2 + c_4 r z + c_5 z^2 + \dots$$

Hence

$$\epsilon_\theta = \frac{u}{r} = c_1 + c_2 \frac{z}{r} + c_3 r + c_4 z + c_5 \frac{z^2}{r} + \dots$$

$$\epsilon_r = \frac{du}{dr} = c_1 + 2c_3 r + c_4 z + 3c_5 r^2 + \dots$$

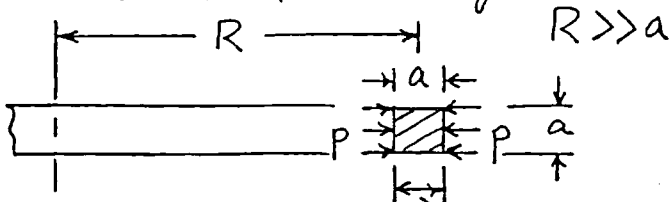
Strains cannot be infinite, so  $c_2=0$ ,  
 $c_5=0, \dots$  Hence at  $r=0$

$$\left. \begin{aligned} \epsilon_\theta &= c_1 + c_4 z + \dots \\ \epsilon_r &= c_1 + c_4 z + \dots \end{aligned} \right\} \text{same}$$

One could also say that directions  
 $r$  and  $\theta$  lack separate meaning at  
 $r=0$ , so  $\epsilon_\theta = \epsilon_r$  there.

14.2-5

Consider  $\epsilon_r$  in a ring.



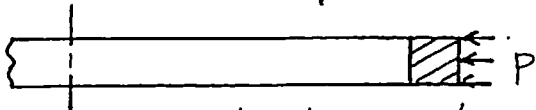
$\delta$  = change in this dimension

$$\delta = \frac{PL}{AE} = \frac{(paRd\theta)a}{(Rd\theta)aE} = \frac{pa}{E}$$

Per radian, the associated stiffness is

$$k_{r1} = \frac{P}{\delta} = \frac{(pa)R}{\delta} = ER$$

Consider  $\epsilon_\theta$  in ring of same dimensions.



$\Delta$  = radial displacement

$$\sigma_\theta = \frac{PR}{a}, \epsilon_\theta = \frac{\sigma_\theta}{E}, \Delta = R\epsilon_\theta = \frac{PR^2}{Ea}$$

Per radian, the associated stiffness is

$$k_{r2} = \frac{P}{\Delta} = \frac{(pa)R}{\Delta} = ER \left(\frac{a}{R}\right)^2$$

$$\text{Stiffness ratio: } \frac{k_{r1}}{k_{r2}} = \left(\frac{R}{a}\right)^2$$

Becomes large if  $R/a$  is large.

14.2-6

(a) Let  $u = a_1 + a_2 r$ . Formal process or trial leads to  $u = \begin{bmatrix} r_2 - r & r - r_1 \\ r_2 - r_1 & r_2 - r_1 \end{bmatrix} \begin{Bmatrix} u_1 \\ u_2 \end{Bmatrix}$

$$\begin{Bmatrix} \epsilon_r \\ \epsilon_\theta \end{Bmatrix} = \begin{Bmatrix} \partial/\partial r \\ 1/r \end{Bmatrix} u = \frac{1}{r_2 - r_1} \underbrace{\begin{bmatrix} -1 & 1 \\ \frac{r_2}{r} - 1 & 1 - \frac{r_1}{r} \end{bmatrix}}_{[B]} \begin{Bmatrix} u_1 \\ u_2 \end{Bmatrix}$$

(b)  $[E] = E \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$ , hence  $[B]^T [E] [B]$  is

$$\frac{E}{(r_2 - r_1)^2} \begin{bmatrix} 2 + \frac{r_2^2}{r^2} - 2 \frac{r_2}{r} & -2 + \frac{r_1 + r_2}{r} - \frac{r_1 r_2}{r^2} \\ \text{symm.} & 2 + \frac{r_1^2}{r^2} - 2 \frac{r_1}{r} \end{bmatrix}$$

$$[k] = \int_0^t \int_0^1 \int_{r_1}^{r_2} [B]^T [E] [B] r dr d\theta dz$$

$$[k] = \frac{E t}{(r_2 - r_1)^2} \begin{bmatrix} -(r_2 - r_1)^2 + r_2^2 \ln \frac{r_2}{r_1} & -r_1 r_2 \ln \frac{r_2}{r_1} \\ -r_1 r_2 \ln \frac{r_2}{r_1} & (r_2 - r_1)^2 + r_1^2 \ln \frac{r_2}{r_1} \end{bmatrix}$$

(c) Let  $r_m = \frac{1}{2}(r_1 + r_2)$ ,  $L = r_2 - r_1$ . With  $r = r_m$ ,

$$[B] = \begin{bmatrix} -1/L & 1/L \\ 1/2r_m & 1/2r_m \end{bmatrix}, [E] = \frac{E}{1-\nu^2} \begin{bmatrix} 1 & \nu \\ \nu & 1 \end{bmatrix}$$

$$[k] = [B]^T [E] [B] t r_m (1) L, \text{ i.e.}$$

$$\frac{E t r_m L}{1-\nu^2} \begin{bmatrix} \frac{1}{L^2} - \frac{\nu}{L r_m} + \frac{1}{4r_m^2} & -\frac{1}{L^2} + \frac{1}{4r_m^2} \\ -\frac{1}{L^2} + \frac{1}{4r_m^2} & \frac{1}{L^2} + \frac{\nu}{L r_m} + \frac{1}{4r_m^2} \end{bmatrix}$$

(d) For  $r_m \rightarrow \infty$ , part (c) yields

$$[k] = \frac{E t r_m}{(1-\nu^2)L} \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \quad (A)$$

In  $[k]$  of part (b),

$$\ln \frac{r_2}{r_1} = \left(\frac{r_2}{r_1} - 1\right) - \frac{1}{2}\left(\frac{r_2}{r_1} - 1\right)^2 + \dots \approx \frac{r_2}{r_1} - 1 = \frac{r_2 - r_1}{r_1} = L/r_1$$

$$[k] = \frac{E t}{L^2} \begin{bmatrix} r_2^2 \frac{L}{r_1} - L^2 & -r_1 r_2 \frac{L}{r_1} \\ -r_1 r_2 \frac{L}{r_1} & r_1^2 \frac{L}{r_1} + L^2 \end{bmatrix} \quad (B)$$

$$\frac{r_2^2}{r_1} L - L^2 = \frac{(r_m + \frac{L}{2})^2}{(r_m - \frac{L}{2})} L - L^2 \approx r_m L - L^2 \approx r_m L$$

$$r_1 r_2 \frac{L}{r_1} = r_2 L \approx r_m L, \quad r_1^2 \frac{L}{r_1} + L^2 = r_1 L + L^2 \approx r_m L$$

Thus (A) & (B) agree if  $\nu = 0$ .

(e) With  $u_1 = 0$ , only  $k_{22}$  remains, which

$$\text{for } r_1 = 0 \text{ is } k_{22} = \frac{E t}{r_2^2} \left[ r_2^2 + \lim_{r_1 \rightarrow 0} \left( r_1^2 \ln \frac{r_2}{r_1} \right) \right]$$

$$\lim_{r_1 \rightarrow 0} \left( r_1^2 \ln \frac{r_2}{r_1} \right) = \lim_{r_1 \rightarrow 0} \frac{\ln r_2 - \ln r_1}{r_1^{-2}} = \lim_{r_1 \rightarrow 0} \frac{-1/r_1}{-2/r_1^3} = 0$$

So  $k_{22}$  reduces to  $E t$ .

$$(f) r_m = \frac{L}{2}, \text{ so } k_{22} = \frac{E t L^2}{2(1-\nu^2)} \left( \frac{1}{L^2} + \frac{2\nu}{L^2} + \frac{1}{L^2} \right)$$

which reduces to  $k_{22} = E t$  if  $\nu = 0$ .

14.2-7

$$\{r_e\} = \int [N]^T \begin{Bmatrix} F_r \\ 0 \end{Bmatrix} dV, \quad F_r = \rho r \omega^2$$

$$\{r_e\} = \frac{\rho \omega^2}{4ab} \int_{-b}^b \int_{-a}^a \int_{-\pi}^{\pi} \begin{Bmatrix} (a-x)(b-y) \\ (a+x)(b-y) \\ (a+x)(b+y) \\ (a-x)(b+y) \end{Bmatrix} r^2 d\theta dx dy$$

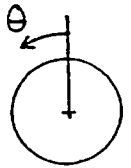
where  $r = r_m + x$ .



14.4-1

(a)

old  $\theta = \text{new } \theta + \frac{\pi}{2}$



$$\begin{aligned} \sin n\left(\theta + \frac{\pi}{2}\right) &= \sin n\theta \cos \frac{n\pi}{2} \\ &\quad + \cos n\theta \sin \frac{n\pi}{2} \\ &= \cos n\theta \sin \frac{n\pi}{2} \end{aligned}$$

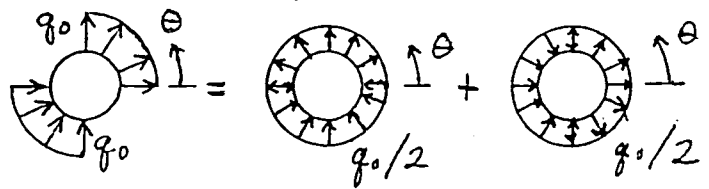
for  $n$  odd

Formula in Fig. 6.6-2 becomes

$$q = \frac{4q_0}{\pi} \left( \cos \theta - \frac{\cos 3\theta}{3} + \frac{\cos 5\theta}{5} \dots \right)$$

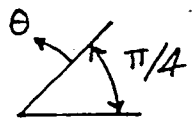
(b)

Fig. 6.6-2 Part (a)



$$\begin{aligned} q &= \frac{2q_0}{\pi} \left( \sin \theta + \frac{\sin 3\theta}{3} + \frac{\sin 5\theta}{5} + \dots \right) \\ &\quad + \frac{2q_0}{\pi} \left( \cos \theta - \frac{\cos 3\theta}{3} + \frac{\cos 5\theta}{5} - \dots \right) \end{aligned}$$

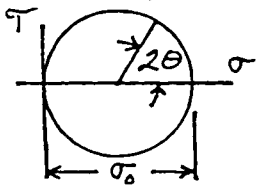
(c) Relocate the  $\theta=0$  position to where  $\theta = \pi/4$  in part (b), i.e.



14.4-2

Must determine tractions  $\Phi$  to be applied to the dashed circular boundary.

Can apply Mohr circle analysis.



$$\sigma_r = \frac{\sigma_0}{2}(1 + \cos 2\theta)$$

$$\tau_{r\theta} = -\frac{\sigma_0}{2}\sin 2\theta$$

(a)  $\sigma_0 = P/ht$ , so

$$\Phi_r = \frac{P}{2ht}(1 + \cos 2\theta), \quad \Phi_\theta = -\frac{P}{2ht}\sin 2\theta, \quad \Phi_z = 0$$

Symmetric terms, zeroth & second harmonics.

Independent of  $r$ .

$$(b) \sigma_0 = -\frac{My}{I} = -\frac{M(r\sin\theta)}{\frac{1}{12}th^3} = -\frac{12M}{th^3}r\sin\theta$$

$$(1 + \cos 2\theta)\sin\theta = (2 - 2\sin^2\theta)\sin\theta \\ = 2(\sin\theta - \sin^3\theta)$$

$$\sin 2\theta \sin\theta = 2\sin\theta \cos\theta \sin\theta \\ = 2\cos\theta(1 - \cos^2\theta)$$

$$\Phi_r = -\frac{12M}{th^3}r\sin\theta \frac{1 + \cos 2\theta}{2} = \frac{12M}{th^3}(-r\sin\theta + r\sin^3\theta)$$

$$\Phi_\theta = -\frac{12M}{th^3}r\sin\theta \left(-\frac{\sin 2\theta}{2}\right) = \frac{12M}{th^3}(r\cos\theta - r\cos^3\theta)$$

$$\Phi_z = 0$$

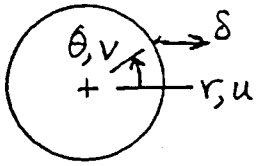
Antisymmetric terms, 1<sup>st</sup> & 3<sup>rd</sup> harmonics.

$\Phi$ 's on boundary of radius  $c$ .

14.4-3

(a)  $u = v = 0, w = w_0$

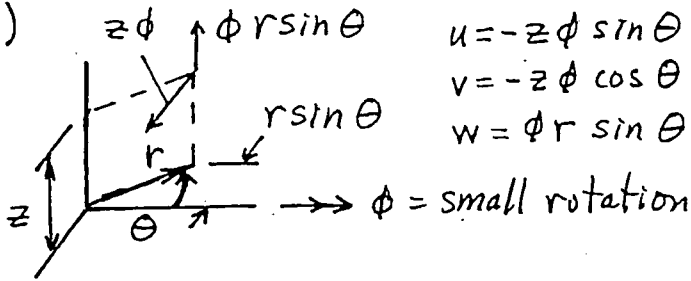
(b)



$$\begin{aligned} u &= \delta \cos \theta \\ v &= -\delta \sin \theta \\ w &= 0 \end{aligned}$$

(c) Orient  $\delta$  vertically in part (b); then  
 $u = \delta \sin \theta, v = \delta \cos \theta, w = 0$

(d)



$$\begin{aligned} u &= -z \phi \sin \theta \\ v &= -z \phi \cos \theta \\ w &= \phi r \sin \theta \end{aligned}$$

14.4-4

(a) Shown by direct substitution.

(b) 1<sup>st</sup> 3 cols., single-barred

2<sup>nd</sup> 3 cols., double-barred

(c) Consider  $a_1$  thru  $a_6$  in turn. Let  $x =$  radial axis along  $\theta = 0$ ,  $y =$  radial axis along  $\theta = \pi/2$ , both in  $z = 0$  plane.

$a_1$  — axial ( $z$ -dir.) translation.

$a_2$  —  $x$  direction translation

$a_3$  — rotation about  $y$  axis.

$a_4$  — rotation about  $z$  axis.

$a_5$  —  $y$  direction translation.

$a_6$  — rotation about  $x$  axis.

14.4-5

(a) Exact, at center  $x = \frac{L}{2}$ :

$$v = \frac{5q_0 L^4}{384EI}, \quad M = -\frac{q_0 L^2}{8}$$

$$v_n = \frac{4q_0 L^4}{n\pi EI n^4 \pi^4} \sin \frac{n\pi x}{L} = \frac{4q_0 L^4}{EI \pi^5 n^5} \sin \frac{n\pi x}{L}$$

$$M_n = EI \frac{d^2 v_n}{dx^2} = -EI \frac{n^2 \pi^2}{L^2} v_n = -\frac{4q_0 L^2}{\pi^3 n^3} \sin \frac{n\pi x}{L}$$

At  $x = \frac{L}{2}$  and with  $n$  odd,

$$v = \sum v_n = \frac{4q_0 L^4}{EI \pi^5} \left(1 - \frac{1}{3^5} + \frac{1}{5^5} - \dots\right)$$

$$M = \sum M_n = \frac{4q_0 L^2}{\pi^3} \left(-1 + \frac{1}{3^3} - \frac{1}{5^3} + \dots\right)$$

Numerical multipliers in ans. are

|       | <u>exact</u> | <u>1 term</u>        | <u>2 terms</u>       | <u>3 terms</u>    |
|-------|--------------|----------------------|----------------------|-------------------|
| $v$   | 0.013021     | 0.013071<br>(+0.38%) | 0.013017<br>(-0.03%) | 0.013021<br>—     |
| $ M $ | 0.125        | 0.1290<br>(+3.2%)    | 0.1242<br>(-0.6%)    | 0.1253<br>(+0.2%) |

(b) Exact, at center  $x = \frac{L}{2}$ :

$$v = \frac{PL^3}{48EI}, \quad M = -\frac{PL}{4}$$

Proceeding as in part (a),

$$v_n = \frac{2PL^3}{EI n^4 \pi^4} \sin \frac{n\pi}{2} \sin \frac{n\pi x}{L}$$

$$\dots \sin \frac{n\pi}{2} \sin \frac{n\pi x}{L}$$

At  $x = \frac{L}{2}$ , only odd terms are nonzero.

$$v = \sum v_n = \frac{2PL^3}{EI \pi^4} \left(1 + \frac{1}{3^4} + \frac{1}{5^4} + \dots\right)$$

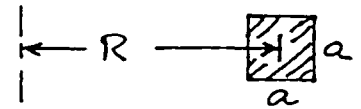
$$M = \sum M_n = \frac{PL}{\pi^2} \left(-1 - \frac{1}{3^2} - \frac{1}{5^2} - \dots\right)$$

Numerical multipliers in ans. are

|       | <u>exact</u> | <u>1 term</u>       | <u>2 terms</u>      | <u>3 terms</u>      |
|-------|--------------|---------------------|---------------------|---------------------|
| $v$   | 0.02083      | 0.02053<br>(-1.45%) | 0.02079<br>(-0.23%) | 0.02082<br>(-0.07%) |
| $ M $ | 0.2500       | 0.2026<br>(-18.9%)  | 0.2252<br>(-9.94%)  | 0.2333<br>(-6.69%)  |

14.5-1

A slender ring of square cross section:



Consider the lowest load harmonic that is statically equivalent to zero:

$$q = q_2^c \cos 2\theta \quad \text{and the associated radial displacement } u = \bar{u}_2 \cos 2\theta$$

Compute  $\bar{u}_2$  by equating strain energy  $U$  to work  $W$  done by load.

$$U = \frac{EI}{2} \int_0^{2\pi} \left[ \frac{d^2 u}{(R d\theta)^2} \right]^2 R d\theta = \frac{E}{2} \frac{a^4}{12} \frac{\bar{u}_2^2}{R^3} 4^2 \int_0^{2\pi} \cos^2 2\theta d\theta$$

$$\text{Substitute } \phi = 2\theta: U = \frac{2Ea^4 \bar{u}_2^2}{3R^3} \int_0^{4\pi} \frac{\cos^2 \phi}{2} d\phi = \frac{2\pi Ea^4 \bar{u}_2^2}{3R^3}$$

$$W = \frac{1}{2} \int_0^{2\pi} u (p a R d\theta) = \frac{q_2^c \bar{u}_2 a R}{2} \int_0^{2\pi} \cos^2 2\theta d\theta$$

$$W = \frac{q_2^c \bar{u}_2 a R}{2} \int_0^{4\pi} \frac{\cos^2 \phi}{2} d\phi = \frac{\pi a R q_2^c \bar{u}_2}{2}$$

$$U = W \quad \text{gives} \quad \bar{u}_2 = \frac{3R^4}{4Ea^3} q_2^c$$

Flexural stiffness per radian might be defined as

$$k_{f2} = \frac{R(q_2^c a)}{\bar{u}_2} = \frac{4ER}{3} \left( \frac{a}{R} \right)^4$$

Circumferential stiffness, from Problem 14.2-5, is

$$k_{r2} = ER \left( \frac{R}{a} \right)^2$$

$$\dots \frac{k_{r2}}{k_{f2}} = \frac{3}{4} \left( \frac{R}{a} \right)^2 \quad \text{Very large if } \frac{R}{a} \text{ is large.}$$

14.5-2

(a) First partition, analogous to that in Eq. 14.5-5, is, with  $[\underline{D}]$  from Eq. 14.4-4,

$$[\underline{D}] \begin{bmatrix} N_{1,r} \sin n\theta & 0 & 0 \\ 0 & -N_{1,z} \cos n\theta & 0 \\ 0 & 0 & N_{1,r} \sin n\theta \end{bmatrix} =$$

$$\begin{bmatrix} N_{1,r} \sin n\theta & 0 & 0 \\ \frac{N_{1,r}}{r} \sin n\theta & \frac{nN_{1,r}}{r} \sin n\theta & 0 \\ 0 & 0 & N_{1,z} \sin n\theta \\ N_{1,z} \cos n\theta & 0 & N_{1,r} \sin n\theta \\ \frac{nN_{1,r}}{r} \cos n\theta & -(N_{1,r} - \frac{N_{1,r}}{r}) \cos n\theta & 0 \\ 0 & -N_{1,z} \cos n\theta & \frac{nN_{1,r}}{r} \cos n\theta \end{bmatrix}$$

Differs from Eq. 14.5-5 in algebraic signs and sine & cosine terms.

(b) To see form of  $[\underline{K}_n]$ , part (a) as compared with that provided by Eq. 14.5-5, need consider only sine & cosine terms. Let  $c = \cos n\theta$ ,  $s = \sin n\theta$ . With  $[\underline{E}]$  populated as in Eq. 14.4-3, part (a) yields

$$\begin{bmatrix} s & s & 0 & s & nc & 0 \\ 0 & ns & 0 & 0 & -c & -c \\ 0 & 0 & s & s & 0 & nc \end{bmatrix} \begin{bmatrix} s & ns & s \\ s & ns & s \\ s & ns & s \\ s & ns & s \\ nc & -c & nc \\ nc & -c & nc \end{bmatrix}$$

Form of  $[\underline{B}]^T [\underline{E}] [\underline{B}] :$

$$\begin{bmatrix} s^2 + nc^2 & n(s^2 - c^2) & s^2 + nc^2 \\ n(s^2 - c^2) & n^2 s^2 + c^2 & n(s^2 - c^2) \\ \dots & n(s^2 - c^2) & s^2 + nc^2 \end{bmatrix}$$

Off-diag. blocks contain  $s_i s_j$  &  $c_i c_j$  with  $i \neq j$ ; integrate to zero over  $\theta = -\pi$  to  $\theta = +\pi$ . On-diag. blocks integrate to  $\pi$  (or to  $2\pi$  for  $n=0$ ), for both  $s^2$  and  $c^2$  terms.

In similar fashion, from the first partition in Eq. 14.5-5,

$$\underbrace{\begin{bmatrix} c & c & 0 & c & -ns & 0 \\ 0 & nc & 0 & 0 & s & s \\ 0 & 0 & c & c & 0 & -ns \end{bmatrix}}_{[\underline{B}]^T} \underbrace{\begin{bmatrix} c & nc & c \\ c & nc & c \\ c & nc & c \\ c & nc & c \\ -ns & s & -ns \\ -ns & s & -ns \end{bmatrix}}_{[\underline{E}] [\underline{B}]}$$

Form of  $[\underline{B}]^T [\underline{E}] [\underline{B}] \rightarrow$

$$\begin{bmatrix} c^2 + n^2 s^2 & n(c^2 - s^2) & c^2 + ns^2 \\ n(c^2 - s^2) & nc^2 + s^2 & n(c^2 - s^2) \\ c^2 + ns^2 & n(c^2 - s^2) & c^2 + ns^2 \end{bmatrix}$$

Since  $s^2$  and  $c^2$  both integrate to  $\pi$  (with limits  $-\pi$  to  $+\pi$ ), the result is the same as produced by part (a).

(c) The middle column of the 6 by 3 result matrix in part (a) changes sign. Then in part (b), row 2 of  $[\underline{B}]^T$  and column 2 of  $[\underline{E}] [\underline{B}]$  change sign. Hence the form of sine and cosine terms in  $[\underline{B}]^T [\underline{E}] [\underline{B}]$  becomes

$$\begin{bmatrix} s^2 + nc^2 & -n(s^2 - c^2) & s^2 + nc^2 \\ -n(s^2 - c^2) & n^2 s^2 + c^2 & -n(s^2 - c^2) \\ s^2 + nc^2 & -n(s^2 - c^2) & s^2 + nc^2 \end{bmatrix}$$

Signs of some terms differ, so  $[\underline{K}_n]$  changes.

14.5-3

(a) If  $n=6$ ,  $\sin n\theta$  goes to  $12\pi$  when  $\theta=2\pi$ , i.e. 12 half-waves, so  $m=3(12)=36$

(b) For solid-of-revolution analysis, 4-noded

$[k] = [B]^T [E] [B]$ . Count multiplications,  $12 \times 12$   $12 \times 6$   $6 \times 6$   $6 \times 12$  ignoring symmetry of  $[k]$ : 4 Gauss pts./el., 20 els., 6 analyses:

$$4 * 20 * 6 * (6 * 12 * 6 + 6 * 12 * 12) = 622,000$$

For 3-D els., 8 node els.,  $[B]$  is  $6 \times 24$ .

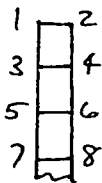
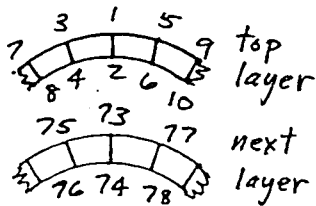
8 Gauss pts./el., 20 \* 36 els.:

$$8 * 20 * 36 (6 * 24 * 6 + 6 * 24 * 24) = 24.9(10)^6$$

$$\text{ratio} = 24.9(10)^6 / 622,000 = 40$$

(c) 3-D mesh numbering:

Solid of rev. mesh numbers:



$$3-D: NB^2 = (72 * 21)(78)^2 = 9.2(10)^6$$

$$\text{Solid of rev.}: NB^2 = 42(4)^2 = 672$$

(exclusive of 3 d.o.f. per node in each case).

With 6 solid-of.-rev. analyses,

$$\text{ratio} = 9.2(10)^6 / (6 * 672) = 2300$$



14.5-4

$$\begin{Bmatrix} u \\ v \end{Bmatrix} = \frac{1}{r_2 - r_1} \begin{bmatrix} (r_2 - r) \cos n\theta & 0 \\ 0 & (r_2 - r) \sin n\theta \\ (r - r_1) \cos n\theta & 0 \\ 0 & (r - r_1) \sin n\theta \end{bmatrix} \begin{Bmatrix} u_1 \\ v_1 \\ u_2 \\ v_2 \end{Bmatrix}$$

From Eq. 10.5-4,

$$\begin{Bmatrix} \epsilon_r \\ \epsilon_\theta \\ \gamma_{r\theta} \end{Bmatrix} = \begin{bmatrix} \frac{\partial}{\partial r} & 0 \\ \frac{1}{r} & \frac{1}{r} \frac{\partial}{\partial \theta} \\ \frac{1}{r} \frac{\partial}{\partial \theta} & \frac{\partial}{\partial r} - \frac{1}{r} \end{bmatrix} \begin{Bmatrix} u \\ v \end{Bmatrix}. \text{ Hence}$$

$$\begin{Bmatrix} \epsilon_r \\ \epsilon_\theta \\ \gamma_{r\theta} \end{Bmatrix} = \frac{1}{r_2 - r_1} \begin{bmatrix} -\cos n\theta & 0 \\ \frac{r_2 - r}{r} \cos n\theta & \frac{r_2 - r}{r} n \cos n\theta \\ -\frac{r_2 - r}{r} n \sin n\theta & -\frac{r_2}{r} \sin n\theta \\ \cos n\theta & 0 \\ \frac{r - r_1}{r} \cos n\theta & \frac{r - r_1}{r} n \cos n\theta \\ -\frac{r - r_1}{r} n \sin n\theta & \frac{r_1}{r} \sin n\theta \end{bmatrix} \begin{Bmatrix} u_1 \\ v_1 \\ u_2 \\ v_2 \end{Bmatrix}$$